



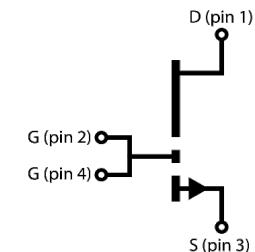
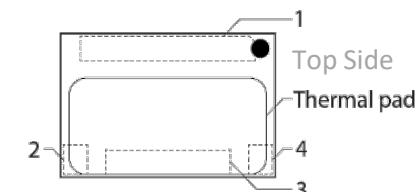
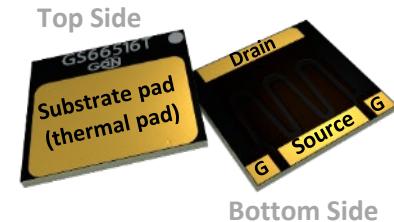
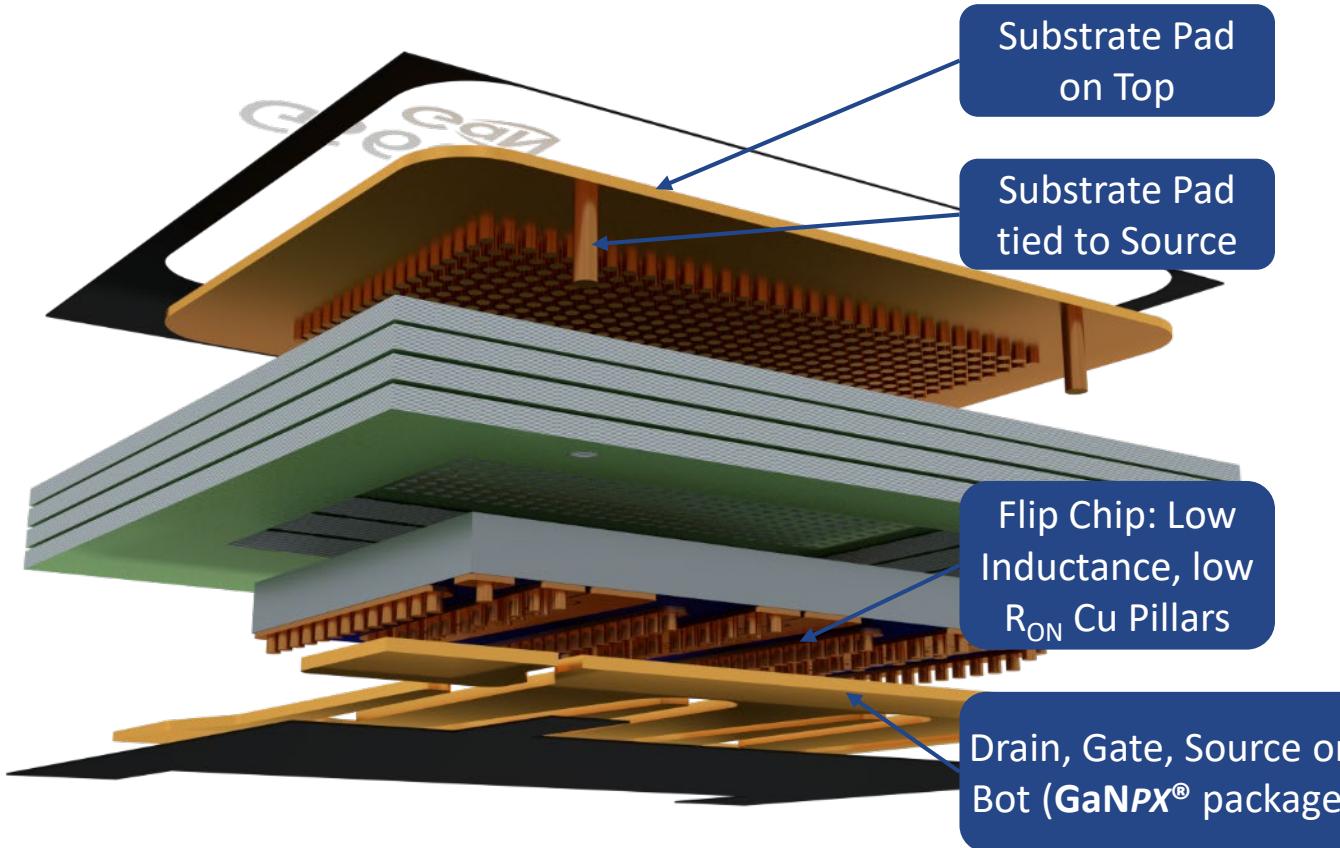
# GN002 Application Note

Thermal Design for GaN Systems' Top-side cooled GaN<sub>PX</sub><sup>®</sup>-T packaged devices

October 30, 2018

- The Basics - Our top side cooled GaNPx®-T package
- Thermal Design for high-power with GaNPx®-T package
- Heatsink Mounting Design Considerations
- Bending Pressure and Deformation Limits

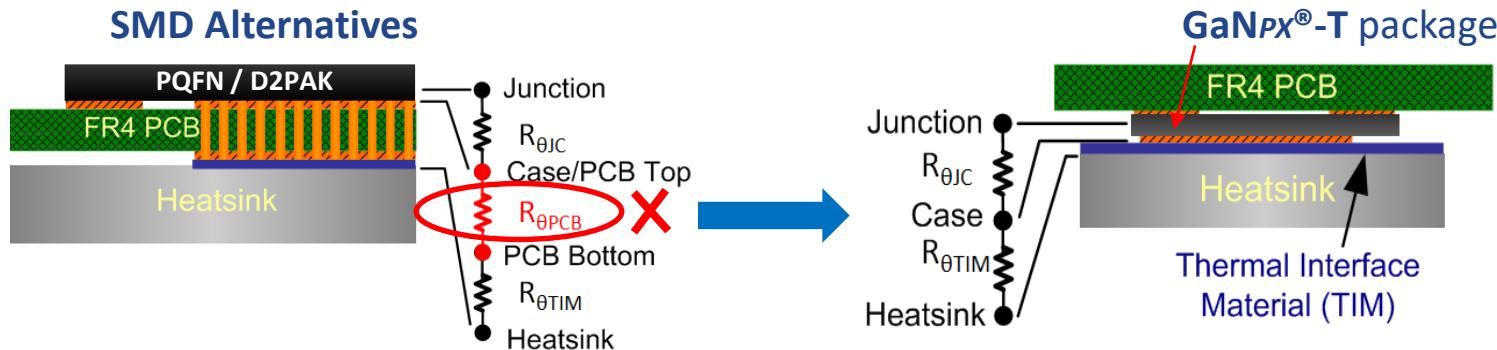
# GaNPx®-T - Embedded Package for Top Side Cooling



GaNPx®-T package, optimized for high power applications with Top-Side Heat Sinking

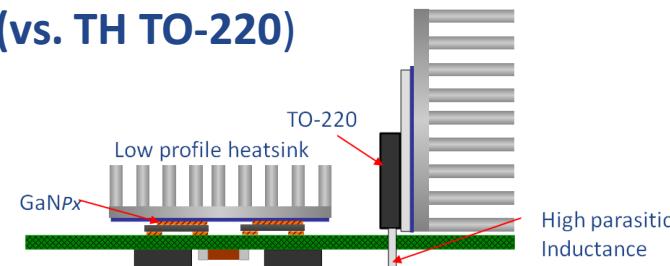
## Eliminates PCB from thermal path (vs. SMD alternatives)

- Simpler PCB layout
- Free up PCB space for improved parasitics
- Better thermal performance



## Enables a more compact, low profile design (vs. TH TO-220)

- High power density, low profile design
- Improved power loop inductance
- Reduced EMI
- Smallest footprint for ultra-high density design

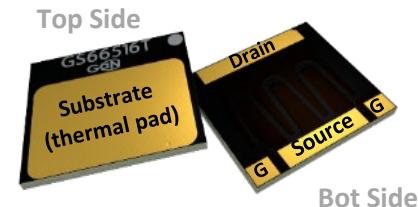


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Heat transfer occurs in three different ways

- **Conduction** – through direct contact
- **Convection** – through fluid movement (air is a fluid)
- **Radiation** – through electromagnetic waves

Our top-side cooled GaNPx®-T packages rely primarily on **conduction** cooling to transfer heat from the internal die surface (junction) to the exterior top and bottom surfaces of the GaNPx®-T package. At a system level convection cooling dominates.



$R_{\theta JC}$

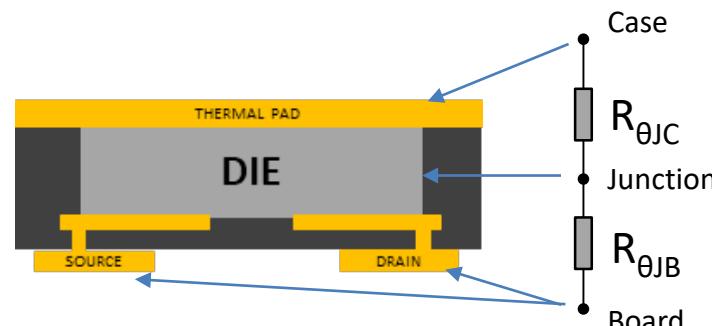
**Junction-to-Case Thermal Resistance**

Thermal Resistance from the Die (junction) to the Substrate pad (case) on the top of the device

$R_{\theta JB}$

**Junction-to-Board Thermal Resistance**

Thermal Resistance from the Die (junction) to the Drain and Source on the bottom of the device (board)



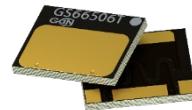
$$R_{\theta JC} = \frac{T_J - T_{Case}}{P}$$

$$R_{\theta JB} = \frac{T_J - T_{Board}}{P}$$

## 650 V Devices

GaNPX® package	$R_{\Theta JC}$ (°C/W)	$R_{\Theta JB}$ (°C/W)
GS66506T	0.7	7.0
GS66508T	0.5	5.0
GS66516T	0.3	3.0

GS66506T



GS66508T



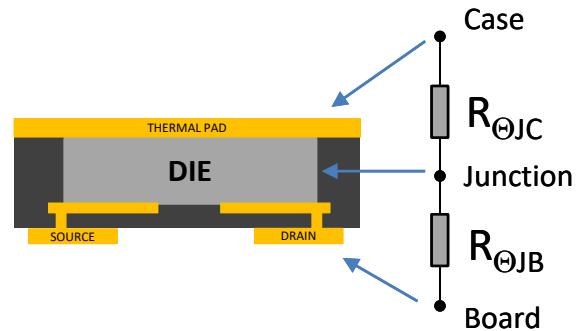
GS66516T



## 100 V Devices

GaNPX® package	$R_{\Theta JC}$ (°C/W)	$R_{\Theta JB}$ (°C/W)
GS61008T	0.55	5.5

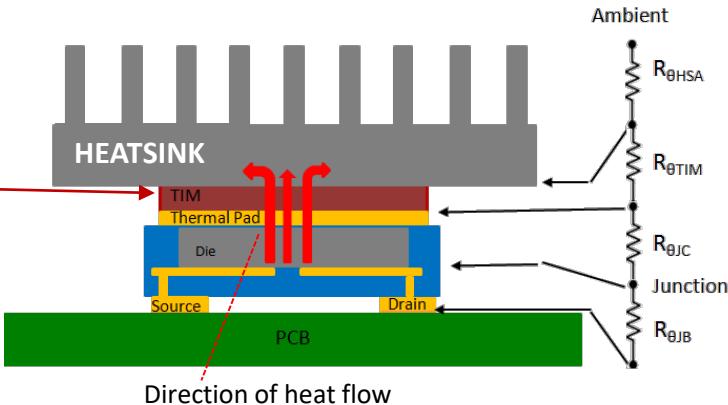
GS61008T



# Using Heatsinks and TIM

The top-side thermal pad provides a path of low thermal resistance for attaching a heatsink.

For improved heat transfer, a **Thermal Interface Material** (TIM) should be placed between the device's thermal pad and the external heatsink. The TIM fills air gaps and voids to improve heat transfer between the device and the heatsink. TIM are available with different thermal resistances.



$$R_{\thetaJA} = R_{\thetaJC} + R_{\thetaTIM} + R_{\thetaHSA}$$

$R_{\thetaTIM}$

TIM Thermal Resistance

#### TIM considerations:

- Thermal Conductivity
- Contact Resistance
- Thickness / Phase
- Electrical Isolation

$R_{\thetaHSA}$

Heatsink-to-Ambient Thermal Resistance

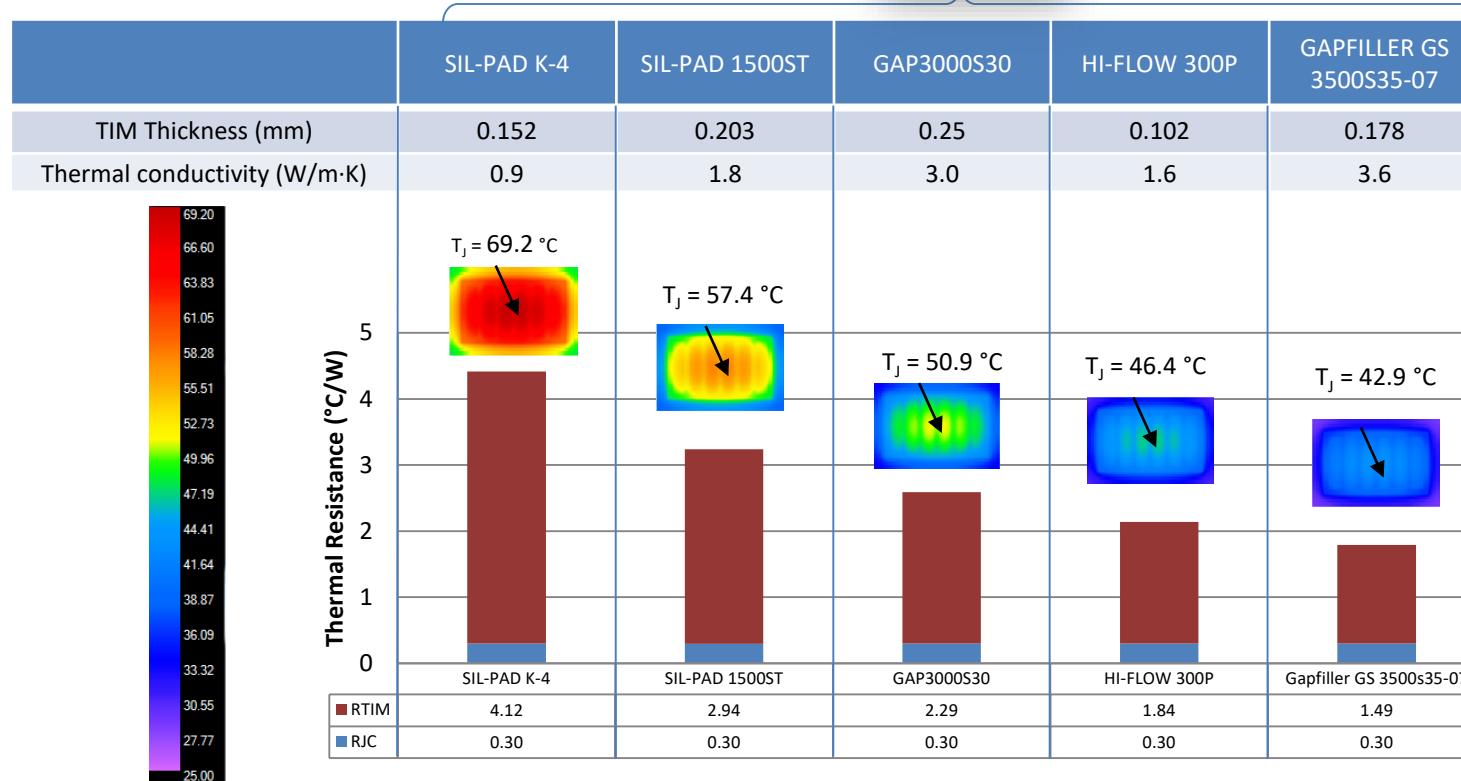
#### Heat Sink considerations

- Thermal Conductivity
- Heatsink size / weight
- Heat Convection path:  
Fin geometry / Air-flow to achieve max efficiency under Zero LFM Air-flow

# GS66516T Thermal Simulation (Typical Design)

## Operating Conditions

- Power = 10 W
- $T_{HS}$  = 25 °C



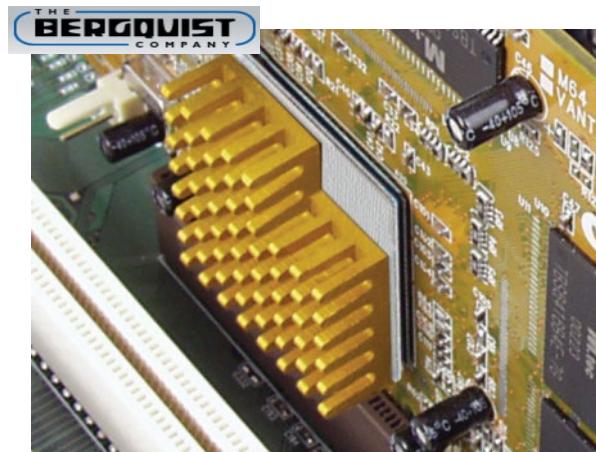
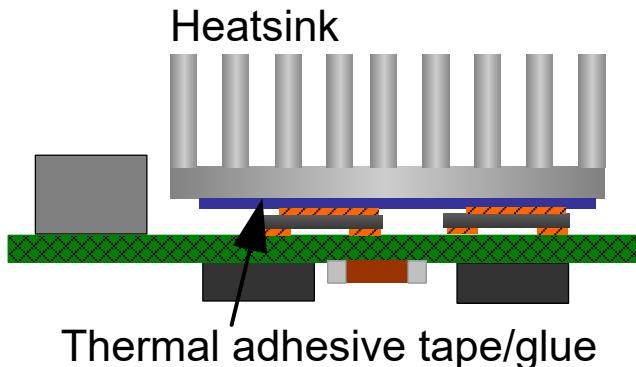
For high-power Electrical Design with GS66516T and PCB (Schematic and Layout), refer to **GN004**

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## Thermal adhesive tape/glue:

For low power design with small lightweight heatsink

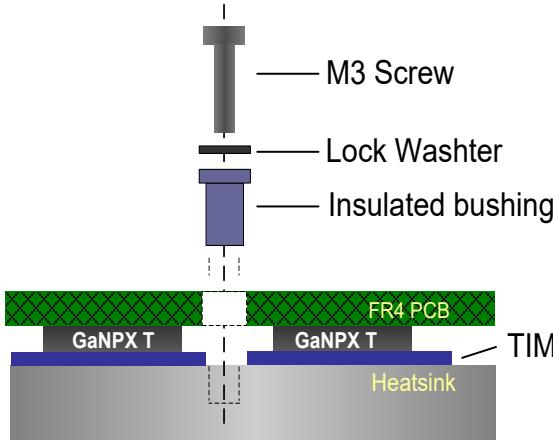
- Low cost
- Simple mechanical design
- No required mounting holes
- Pre-applied pressure during assembly
- Heatsink floating or grounded via clip for EMI



Example: Bergquist® BondPly series 100

## Center mounting hole

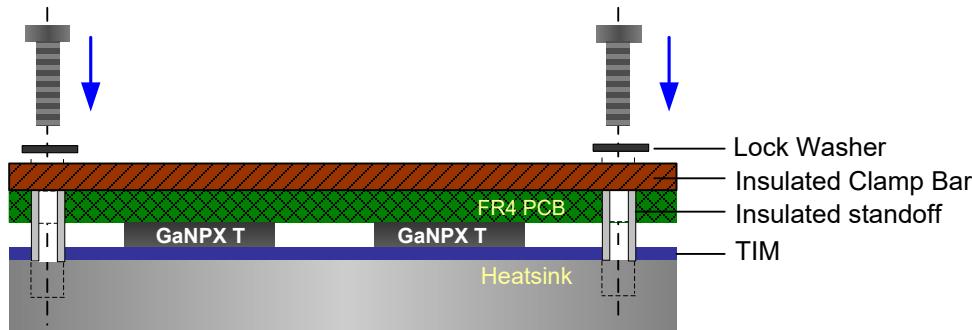
- Balanced pressure across 2 devices
- Typical recommended maximum pressure ~50psi.
- Tested up to 100psi without failure
- Suitable for small heatsink attachment



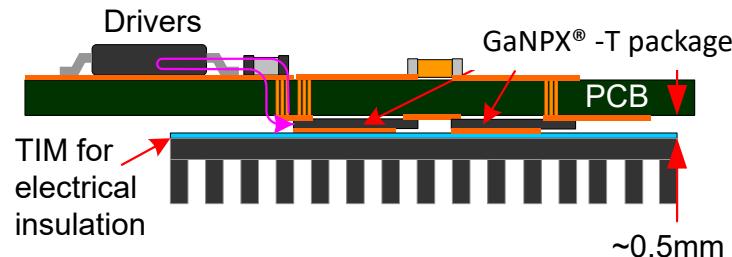
## 2 or more mounting holes for large heatsink

More susceptible to PCB bending stress:

- Excess PCB bending causes stress to GaNPX®-T package and other SMD parts which should be avoided
- Locate mounting holes close to GaNPX®-T package
- Recommended to use a supporting clamp bar on top of PCB for additional mechanical support



## GaNPx®-T package on opposite side to other components



Heatsink/chassis mechanically attached to GaNPx®-T package

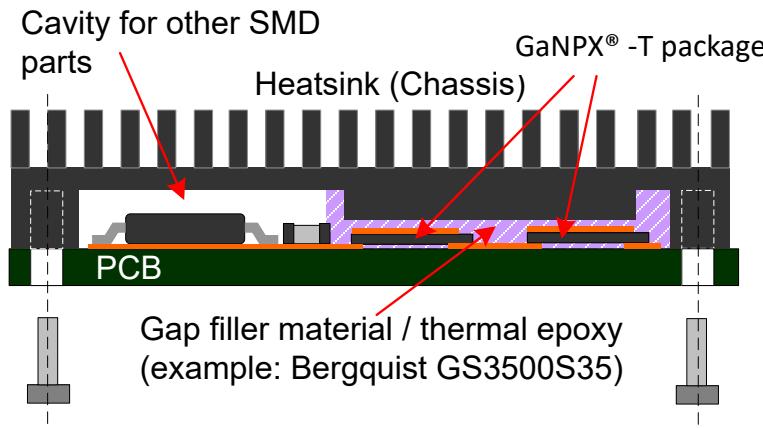
### Pros

- Good thermal performance
- Simple heatsink design

### Cons

- Mechanical stress
- Creepage distances
- Longer gate drive loop

## GaNPx®-T package on same side as other components



Heatsink mechanically attached to PCB

Bottom of heatsink contoured to define the gap and accommodate other parts

Gap filled with gap filler or thermal epoxy.

### Pros

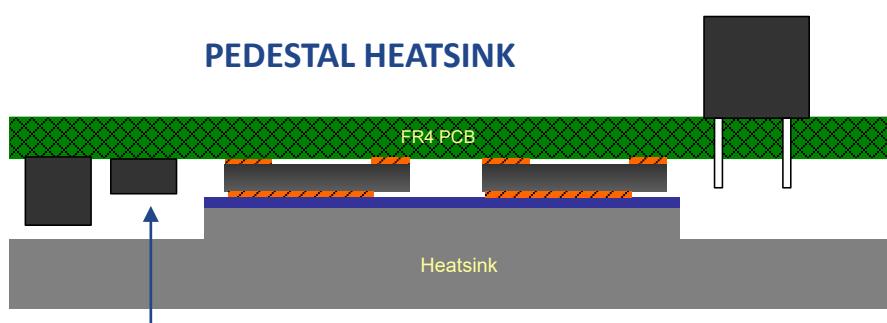
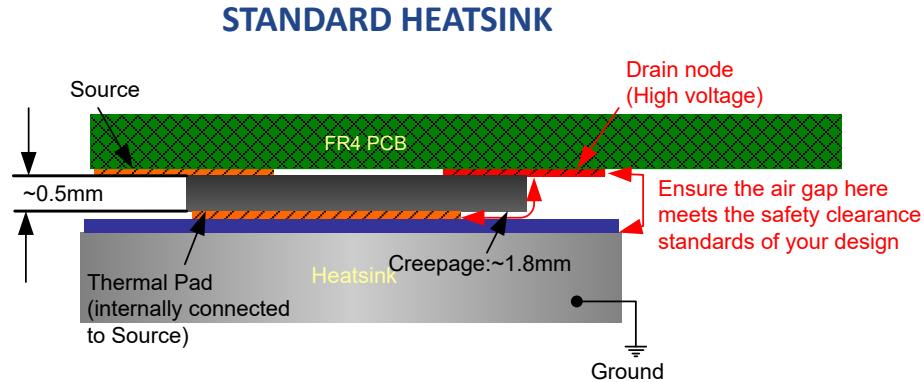
- No direct mechanical stress to GaNPx®-T package
- Single side placement
- Tight gate drive layout

### Cons

- Higher thermal resistance
- Complicated heatsink design

## When using a heatsink, design to meet the Regulatory creepage and clearance requirements

- Use TIM to cover Heatsink edge in areas where clearances must meet Standards
- Avoid placing Through Hole Components near GaN<sup>®</sup> -T package
- Use **Pedestal Heatsink** design to increase clearances and allow for placement of SMT components under the heatsink

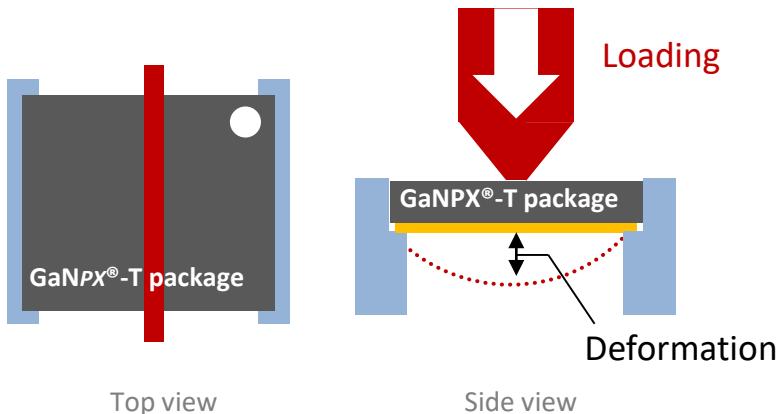


A pedestal heatsink provides clearance beneath the heatsink for the placement of SMT devices

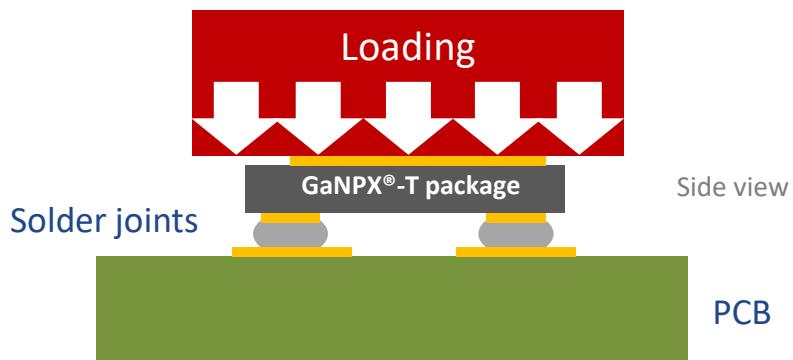
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Part Number	Deformation Safe Limit ( $\mu\text{m}$ )	Pressure Safe Limit (PSI)
GS66508T	50	100
GS66516T	120	100

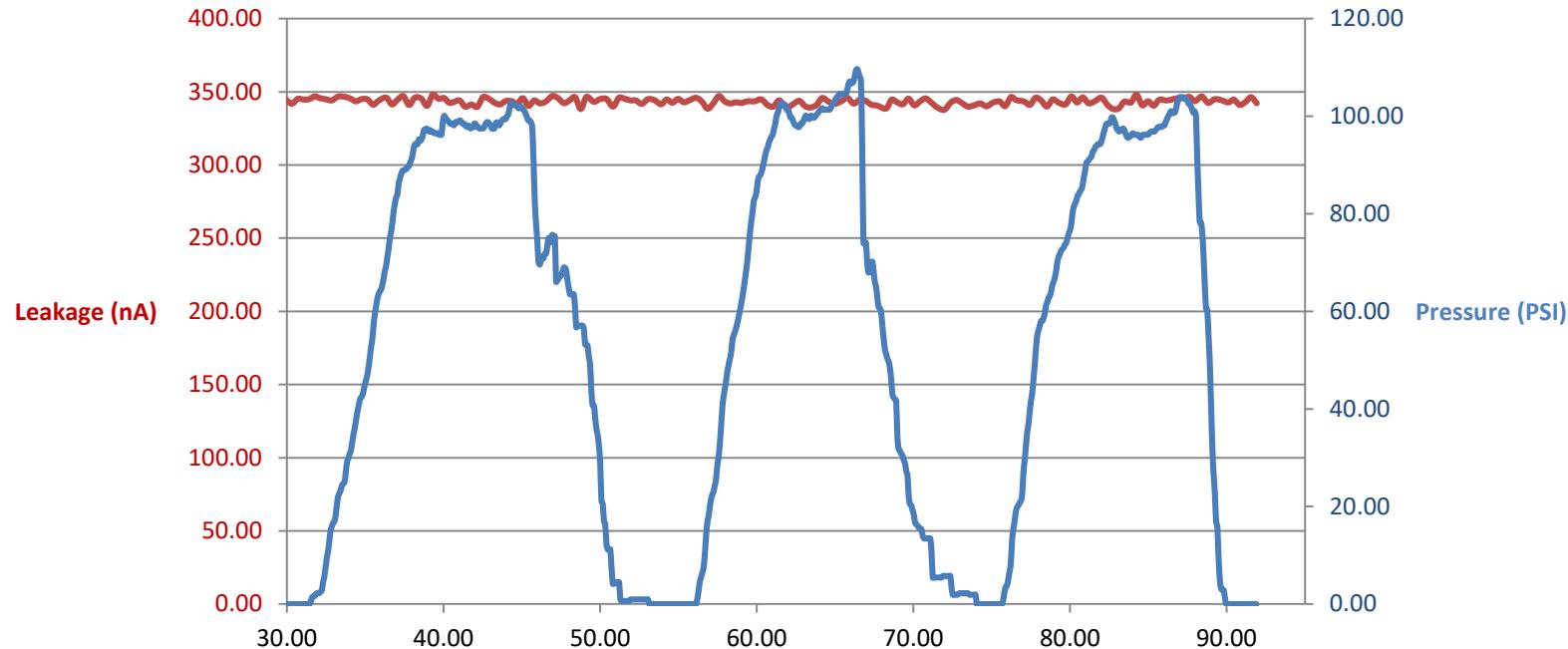
## Deformation Test



## Pressure Test



## Example: GS66508T



DUT subject to 100 PSI over 3 pulses, with no shift in Leakage Currents

400 volts  $V_{DS}$  applied to each DUT (@ 25°C)

Leakage Current =  $I_{DSS} + I_{GS} + I_{BULK}^*$  (\*Substrate)

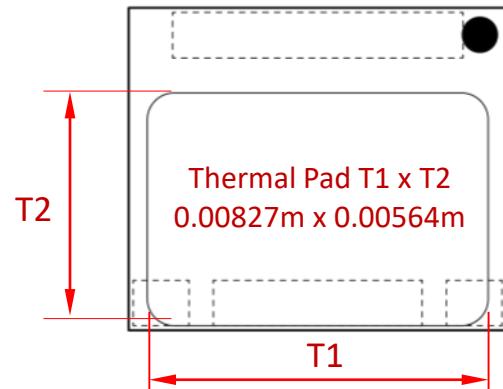
The contact area of the thermal pads must be calculated.  
In this example the total contact area value of both devices is  $0.0000933 \text{ m}^2$ .

Also required are the properties of the fastener itself. In this case a M3 x 0.5 steel screw

- Thread diameter = 0.003 m
- 75% of proof loading = 847.5 N

Values for other fasteners can be found by referencing the following ISO standards

- ISO 898-1:2013
- ISO 898-7:1992



# Torque vs. Pressure relationship

With these values we can now use the following formulas to plot the relationship between fastener torque and the pressure exerted on the devices in this example

$$P_{i(i=0,1;0.1)} = \frac{F_{i(i=0,1;0.1)}}{A}$$

$$F_{i(i=0,1;0.1)} = \frac{Q_{i(i=0,1;0.1)}}{\beta \times \gamma \times d}$$

Q = fastener torque (N-m)

P = pressure on device (kPa)

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A = contact area of thermal pad(s) (m<sup>2</sup>)

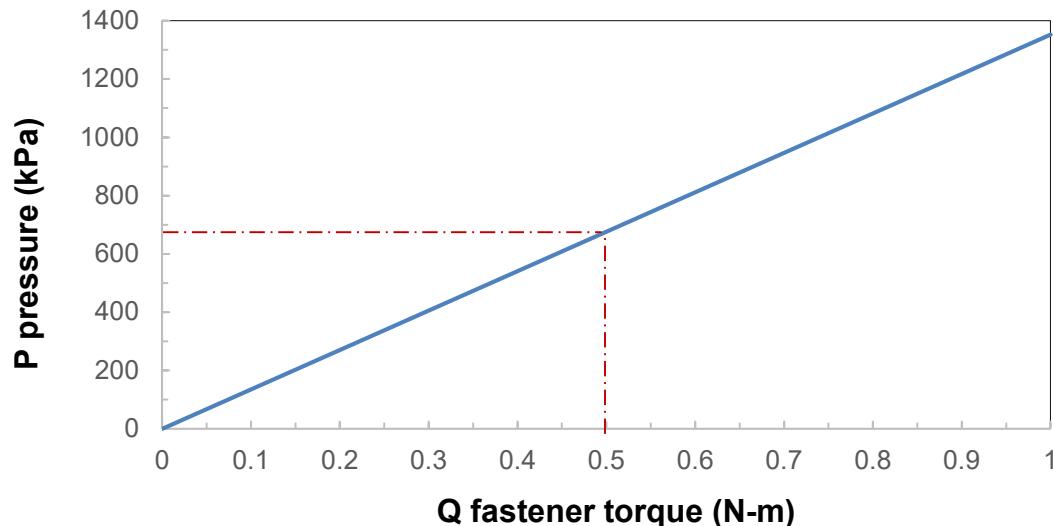
d = screw diameter (m)

F = 75% ISO proof loading (N)

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$\beta = 0.2$  (threads factor)

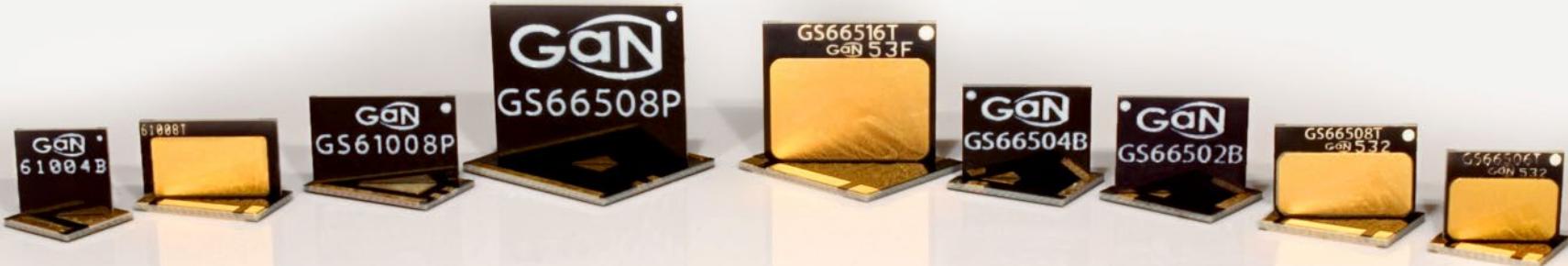
$\gamma = 0.115$  (PCB assembly factor)



A torque of **0.5 N-m** generates a pressure of  $\sim 680$  kPa (98.6 PSI) on the thermal pads of the GS66516T devices, the published maximum.

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